



مدرس خصوصى



اونلاين

يحصل الطالب على ...

* مقاطع فيديو هات لشرح المقرر بشكل وافي

* ملخص للمادة للمراجعة النهائية

* محاضرات مباشرة على برنامج Zoom

* تواصل مستمر مع معلم المادة للمناقشة



للتواصل

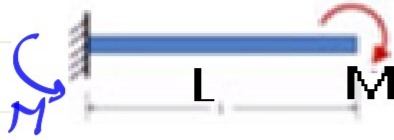


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①



$$\begin{array}{c} m \quad \boxed{-} \quad m \\ \text{B.M.D} \\ \text{---} \quad \ominus \quad \text{---} \\ \text{S.F.D} \end{array}$$

boundary conditions

at $x=0 \Rightarrow \theta=0, \Delta=0$

$$\theta = \frac{1}{EI} \int M dx = -\frac{M}{EI} x + C_1$$

$$\begin{aligned} \Delta &= \frac{1}{EI} \iint M dx = \frac{1}{EI} \int (-Mx + C_1) dx \\ &= \frac{1}{EI} \left[-M \frac{x^2}{2} + C_1 x + C_2 \right] \end{aligned}$$

at $x=0$

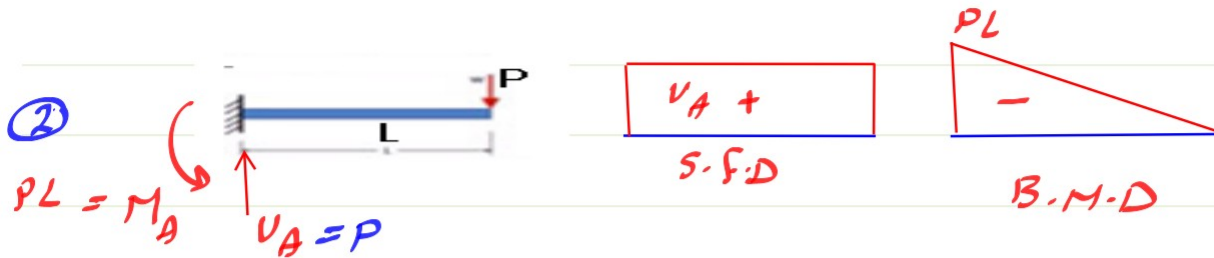
$$\theta = -\frac{M}{EI} (0) + C_1 = 0 \Rightarrow C_1 = 0$$

$$\theta = -\frac{M}{EI} x \Rightarrow \theta_B = -\frac{M}{EI} L$$

$$\Delta = \frac{1}{EI} \left[-M \left(\frac{0}{2} \right) + 0 + C_2 \right] = 0 \Rightarrow C_2 = 0$$

$$\Delta = -M \frac{x^2}{2}, \quad \max \Delta = -\frac{ML^2}{2}$$

Max value of $M = M$



$$M = V_A x - M_A = Px - PL$$

$$\theta = \frac{1}{EI} \int M dx = \frac{1}{EI} \int (Px - PL) dx$$

$$\theta = \frac{1}{EI} \left(P \frac{x^2}{2} - PLx + C_1 \right)$$

$$\Delta = \frac{1}{EI} \int \theta dx = \frac{1}{EI} \int \left(P \frac{x^2}{2} - PLx + C_1 \right) dx$$

$$\Delta = \frac{1}{EI} \left[P \frac{x^3}{6} - PL \frac{x^2}{2} + C_1 x + C_2 \right]$$

$$\text{at } x = 0 \Rightarrow \theta = 0, \Delta = 0$$

$$\theta = P(0) - PL(0) + C_1 = 0 \Rightarrow C_1 = 0$$

$$\Delta = P(0) - PL(0) + C_1(0) + C_2 = 0 \Rightarrow C_2 = 0$$

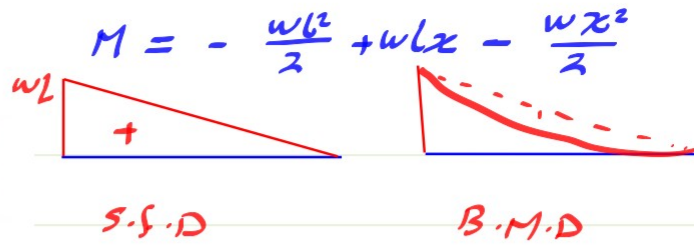
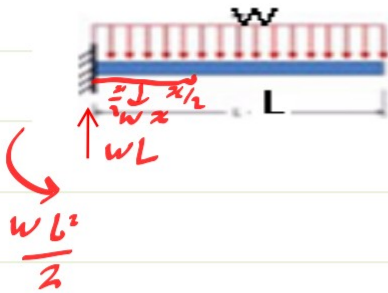
$$\theta = P \frac{x^2}{2} - PLx \Rightarrow \theta_B = P \frac{L^2}{2} - PL^2 = - \frac{PL^2}{2EI}$$

$$\Delta = \frac{1}{EI} \left(P \frac{x^3}{6} - PL \frac{x^2}{2} \right)$$

$$\Delta_B = \frac{1}{EI} \left(\frac{PL^3}{6} - PL \frac{L^2}{2} \right) = - \frac{PL^3}{3EI}$$

$$\text{max value of B.M} = PL$$

③



$$\theta = \frac{1}{EI} \int M dx = \frac{1}{EI} \int \left(-\frac{wL^2}{2} + wLx - \frac{wx^2}{2} \right) dx$$

$$\theta = \frac{1}{EI} \left(-\frac{wL^2}{2} x + wL \frac{x^2}{2} - \frac{wx^3}{6} + C_1 \right)$$

$$\Delta = \frac{1}{EI} \int \theta dx = \frac{1}{EI} \left[-\frac{wL^2}{2} \cdot \frac{x^2}{2} + wL \frac{x^3}{6} - \frac{wx^4}{24} + C_1 x + C_2 \right]$$

$$\text{At } x=0 \Rightarrow \theta=0, \Delta=0 \Rightarrow C_1 = C_2 = 0$$

$$\theta = \frac{1}{EI} \left(-\frac{wL^2}{2} x + wL \frac{x^2}{2} - \frac{wx^3}{6} \right)$$

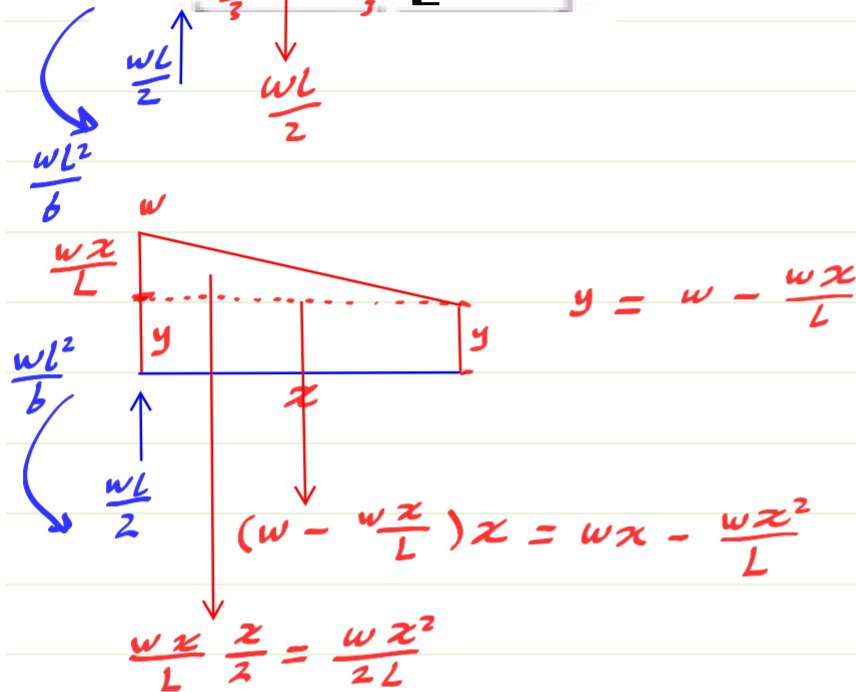
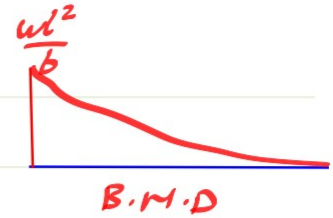
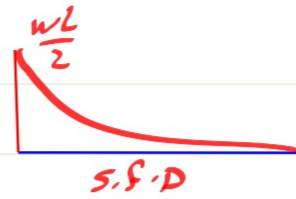
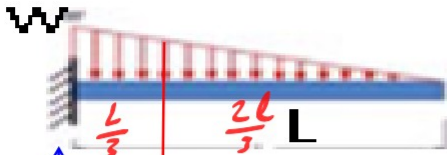
$$\theta_B = \frac{1}{EI} \left(-\frac{wL^3}{2} + wL \frac{L^2}{2} - \frac{wL^3}{6} \right) = -\frac{wL^3}{6EI}$$

$$\Delta = \frac{-wL^2 x^2}{4} + \frac{wL x^3}{6} - \frac{wx^4}{24}$$

$$\Delta_B = \frac{-wL^4}{4} + \frac{wL^4}{6} - \frac{wL^4}{24} = -\frac{wL^4}{8EI}$$

$$\text{Max value of B.M} = \frac{wL^2}{2}$$

4)



$$M = -\frac{wL^2}{6} + \frac{wL}{2}x - \frac{wx^2}{2L} \cdot \frac{2}{3}x - (wx - \frac{wx^2}{L})\left(\frac{x}{2}\right)$$

$$M = -\frac{wL^2}{6} + \frac{wL}{2}x - \frac{wx^3}{3L} - \frac{wx^2}{2} + \frac{wx^3}{2L}$$

$$\theta = \frac{1}{EI} \left[-\frac{wL^2}{6}x + \frac{wL}{2} \cdot \frac{x^2}{2} - \frac{w}{3L} \cdot \frac{x^4}{4} - \frac{w}{2} \cdot \frac{x^3}{3} + \frac{w}{2L} \cdot \frac{x^4}{4} \right] + C_1$$

$$\Delta = -\frac{wL^2}{6} \cdot \frac{x}{2} + \frac{wL}{4} \cdot \frac{x^3}{3} - \frac{w}{12L} \cdot \frac{x^5}{5} - \frac{w}{6} \cdot \frac{x^4}{4} + \frac{w}{8L} \cdot \frac{x^5}{5} + C_1x + C_2$$

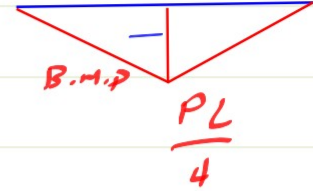
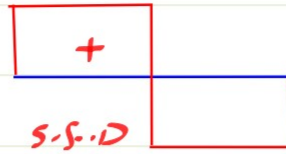
$$\text{at } x=0 \Rightarrow \Delta=0, \theta=0 \Rightarrow C_1=C_2=0$$

$$\theta_B = -\frac{wL^3}{24}$$

$$\Delta_B = -\frac{wL^4}{30}, \text{ max Bending} = \frac{wL^2}{6}$$

$$M = \frac{P}{2}x$$

$\frac{P}{2}$



$$\theta = \frac{1}{EI} \int M dx = \frac{1}{EI} \left(\frac{Px^2}{4} + C_1 \right)$$

$$\Delta = \frac{1}{EI} \int \theta dx = \frac{1}{EI} \left[\frac{Px^3}{12} + C_1x + C_2 \right]$$

$$\text{at } x=0 \Rightarrow \Delta=0 \quad \text{at } x=\frac{L}{2} \Rightarrow \theta=0$$

$$\frac{P}{4} \left(\frac{L}{2} \right)^2 + C_1 = 0 \Rightarrow C_1 = -\frac{PL^2}{16}$$

$$\theta = \frac{1}{EI} \left(\frac{Px^2}{4} - \frac{PL^2}{16} \right) \Rightarrow \theta_A = -\frac{PL^2}{16}$$

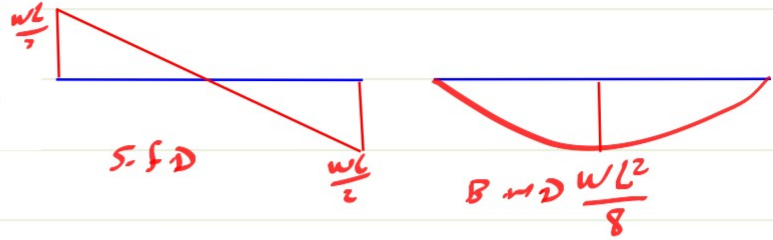
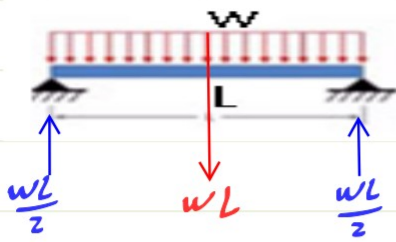
$$\Delta = \frac{P}{12} (0)^3 + C_1(0) + C_2 = 0 \Rightarrow C_2 = 0$$

$$\Delta = \frac{1}{EI} \left(\frac{Px^3}{12} - \frac{PL^2}{16} \cdot x \right)$$

$$\Delta_C = \frac{1}{EI} \left[\frac{P}{12} \left(\frac{L}{2} \right)^3 - \frac{PL^2}{16} \cdot \frac{L}{2} \right] = -\frac{PL^3}{48EI}$$

$$M = \frac{wL}{2}x - \frac{wx^2}{2}$$

⑥



$$\theta = \frac{1}{EI} \int M dx = \frac{1}{EI} \left[\frac{wL}{2} \cdot \frac{x^2}{2} - \frac{w}{2} \cdot \frac{x^3}{3} + C_1 \right]$$

$$\Delta = \frac{1}{EI} \int \theta dx = \frac{1}{EI} \left[\frac{wL}{2} \cdot \frac{x^3}{6} - \frac{w}{2} \cdot \frac{x^4}{12} + C_1 x + C_2 \right]$$

$$\text{at } x=0 \Rightarrow \Delta=0, \text{ at } x=\frac{L}{2} \Rightarrow \theta=0$$

$$\theta = \frac{wL}{4} \left(\frac{L}{2}\right)^2 - \frac{w}{6} \left(\frac{L}{2}\right)^3 + C_1 = 0 \Rightarrow C_1 = -\frac{wL^3}{24}$$

$$\theta = \frac{wL}{4} x^2 - \frac{w}{6} x^3 - \frac{wL^3}{24}$$

$$\text{at } x=0 \Rightarrow \theta = -\frac{wL^3}{24}$$

$$\Delta = \frac{wL}{2} (0) - \frac{w}{2} (0) + C_1 (0) + C_2 = 0 \Rightarrow C_2 = 0$$

$$\Delta = \frac{1}{EI} \left[\frac{wL}{12} x^3 - \frac{w}{24} x^4 - \frac{wL^3}{24} x \right]$$

$$\text{at } x = \frac{L}{2}$$

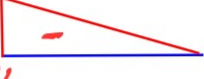
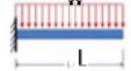
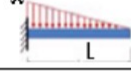
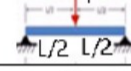
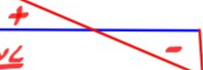
$$\Delta = \frac{1}{EI} \left[\frac{wL}{12} \left(\frac{L}{2}\right)^3 - \frac{w}{24} \left(\frac{L}{2}\right)^4 - \frac{wL^3}{24} \left(\frac{L}{2}\right) \right] = -\frac{5wL^4}{384EI}$$

$$\text{maximum bending} = \frac{wL^2}{8}$$

Assignment No (2)

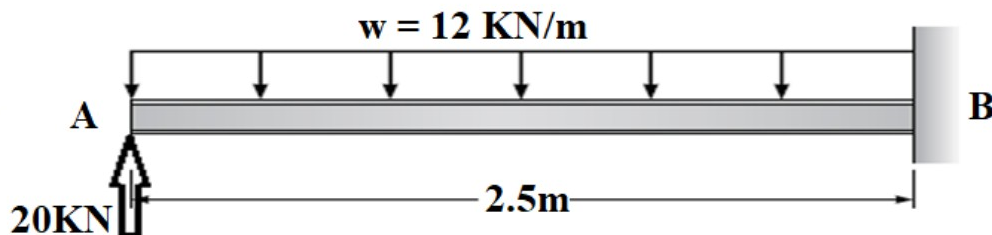
Deflection of Beams - Geometric Methods

1. Complete the table below, take $M=14\text{KN.m}$, $L=2.1\text{m}$, $w=15\text{KN/m}$ and $P=26\text{KN}$

No	Type of beam	Bending moment diagram	Shear force diagram	Value of Slope	Maximum Value of Deflection
1				$\theta = \frac{-ML}{EI} = \frac{-29.4}{EI}$	$\Delta = \frac{-ML^2}{2EI} = \frac{-30.87}{EI}$
2				$\theta = \frac{-PL^2}{2EI} = \frac{-57.33}{EI}$	$\Delta = \frac{-PL^3}{3EI} = \frac{-80.76}{EI}$
3				$\theta = \frac{-wL^3}{6EI} = \frac{-23.15}{EI}$	$\Delta = \frac{-wL^4}{8EI} = \frac{-36.46}{EI}$
4				$\theta = \frac{-wL^3}{24EI} = \frac{-5.78}{EI}$	$\Delta = \frac{-wL^4}{306EI} = \frac{-9.72}{EI}$
5				$\theta = \frac{-PL^2}{16EI} = \frac{-7.16}{EI}$	$\Delta = \frac{-PL^3}{48EI} = \frac{-5}{EI}$
6				$\theta = \frac{-wL^3}{24EI} = \frac{-5.78}{EI}$	$\Delta = \frac{-5wL^4}{384EI} = \frac{-3.8}{EI}$

$$\frac{wL^2}{8}$$

2. A propped cantilever beam carries a uniformly distributed load of 20 kN/m over its entire length, as shown in figure below. Using the geometric methods, determine the slopes at A, B and the deflection at A.



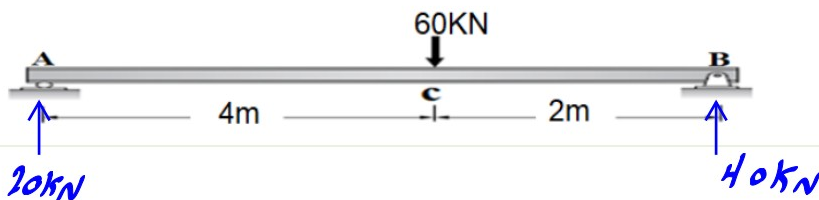
$$\theta_A = \frac{wL^3}{6EI} - \frac{PL^2}{2EI} = \frac{12 \times 2.5^3}{6EI} - \frac{20 \times 2.5^2}{2EI} = \frac{-125}{4EI}$$

$$\theta_B = \text{zero}$$

✓

$$\Delta_A = \frac{PL^3}{3EI} - \frac{wL^4}{8EI} = \frac{20 \times 2.5^3}{3EI} - \frac{12 \times 2.5^4}{8EI} = \frac{45.57}{EI}$$

3. For the steel beam in figure below, compute the end slope at supports A & B. Also determine the location and value of the maximum deflection.



$$\Delta_A = 0$$

$$\Delta_B = 0$$

$$M = 20x - 60(x - 4)$$

$$\theta = \frac{1}{EI} \int M dx = \frac{1}{EI} \left[10x^2 - 30(x - 4)^2 + C_1 \right]$$

$$\Delta = \frac{1}{EI} \int \theta dx = \frac{1}{EI} \left[\frac{10}{3} x^3 - 10(x - 4)^3 + C_1 x + C_2 \right]$$

$$\text{at } x = 0 \Rightarrow \Delta = 0 \Rightarrow C_2 = 0$$

$$\text{at } x = 6 \Rightarrow \Delta = 0$$

$$0 = \frac{10}{3} \times 6^3 - 10(2)^3 + C_1 \times 6$$

$$C_1 = -106.67$$

$$\theta = \frac{1}{EI} (10x^2 - 30(x-4)^2 - 106.67)$$

$$\Delta = \frac{1}{EI} \left(\frac{10}{3} x^3 - 10(x-4)^3 - 106.67x \right)$$

$$\theta_A = \frac{-106.67}{EI}$$

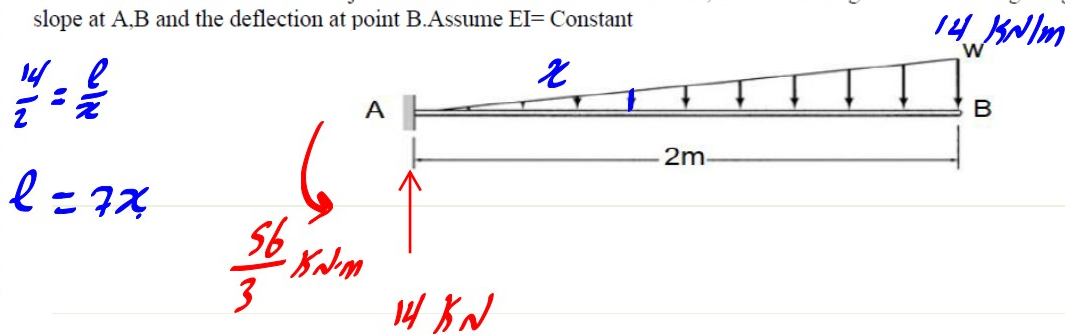
$$\theta_B = \frac{1}{EI} (10 \times 6^2 - 30(2)^2 - 106.67) = \frac{133}{EI}$$

✓ for max deflection

$$10x^2 - 106.67 = 0 \Rightarrow x = 3.26 \text{ m}$$

$$\begin{aligned}
 \Delta_{max} &= \frac{1}{EI} \left(\frac{10}{3} \times 3.26^3 - 106.67 \times 3.26 \right) \\
 &= \frac{-232.26}{EI}
 \end{aligned}$$

4. The RC Cantilever beam is subjected to distributed load $w = 14 \text{ kN/m}$, as shown in figure below. Using the geometric method, find the slope at A.B and the deflection at point B. Assume $EI = \text{Constant}$



$$M = -\frac{56}{3} + 14x - 0.5x * 7x * \frac{1}{3}x$$

$$M = -\frac{56}{3} + 14x - \frac{7}{6}x^3$$

$$\theta = \int M dx = \frac{1}{EI} \left(-\frac{56}{3}x + 7x^2 - \frac{7}{24}x^4 + C_1 \right)$$

$$\text{at } x = 0 \Rightarrow \theta = 0 \Rightarrow C_1 = 0$$

$$\theta = -\frac{56}{3}x + 7x^2 - \frac{7}{24}x^4$$

$$\theta_B = \frac{-14}{EI}$$

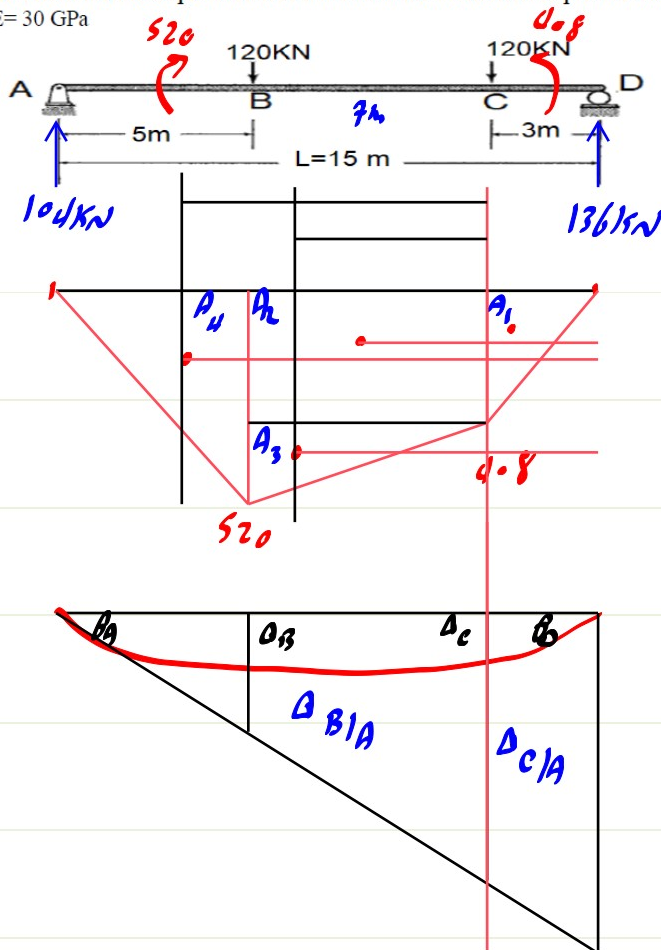
$$\Delta = \frac{1}{EI} \int \theta dx = \frac{1}{EI} \left[-\frac{28}{3}x^2 + \frac{7}{3}x^3 - \frac{7x^5}{120} + C_2 \right]$$

$$\text{at } x = 0 \Rightarrow \Delta = 0 \Rightarrow C_2 = 0$$

$$\Delta_B = -\frac{28}{3} * 2^2 + \frac{7}{3} * 2^3 - \frac{7 * 2^5}{120} = -\frac{20.54}{EI}$$

$$\frac{\Delta}{EI} = \theta_A - \theta_D$$

5. Use the moment-area method to determine the slopes at ends A and D and the deflections at points B and C of the beam shown in Fig. below. Take $EI = \text{constant}$ and $E = 30 \text{ GPa}$



$$A_1 = 612$$

$$A_2 = 2856$$

$$A_3 = 392$$

$$A_4 = 1300$$

$$\theta_A = \frac{\Delta_{D/A}}{15}$$

$$\Delta_{D/A} EI = 612 \times 2 + 2856 \times 6.5 + 392 \times 7.67 + 1300 \times 11.67$$

$$\Delta_{D/A} = \frac{37960}{EI}$$

$$\theta_A = \frac{37960}{15EI} = \frac{2530.67}{30 \times 10^6 I} = \frac{8.43 \times 10^{-5}}{I}$$

$$\frac{\Delta_{D/A}}{15} = \frac{\Delta_B + \Delta_{B/A}}{5}$$

$$\Delta_{B/A} = 1300 \times \frac{S}{3} = 2166.67$$

$$\frac{37960}{15} = \frac{\Delta_B + 2166.67}{5}$$

$$\Delta_B = \frac{10486}{30 \times 106}$$

$$\frac{\Delta_{D/A}}{15} = \frac{\Delta_C + \Delta_{C/A}}{12}$$

✓

$$EI \Delta_{C/A} = 2856 \times 3.5 + 392 \times \frac{2}{3} \times 7 + 1300 \times 8.67$$

$$\Delta_{C/A} = \frac{23356}{EI}$$

$$\frac{37960}{15} = \frac{\Delta_C + 23096.6}{12}$$

$$\Delta_C = \frac{7271}{EI}$$

$$\theta_A - \theta_D = \frac{A}{EI}$$

$$\frac{2530.67 - (6/12 + 2856 + 392 + 1300)}{EI} = \theta_D$$

$$\theta_D = \frac{-2629.33}{EI}$$